

12. External bearing sealing devices

External seals have two main functions: to prevent lubricating oil from leaking out, and, to prevent dust, water, and other contaminants from entering the bearing. When selecting a seal, the following factors need to be taken into consideration: the type of lubricant (oil or grease), seal peripheral speed, shaft fitting errors, space limitations, seal friction and resultant heat increase, and cost.

Sealing devices for rolling bearings fall into two main classifications: non-contact seals and contact seals.

- **Non-contact seals:** Non-contact seals utilize a small clearance between the shaft and the housing cover. Therefore friction is negligible, making them suitable for high speed applications.

In order to improve sealing capability, clearance spaces are often filled with lubricant.

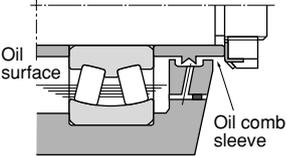
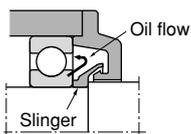
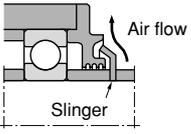
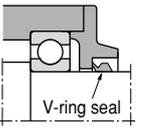
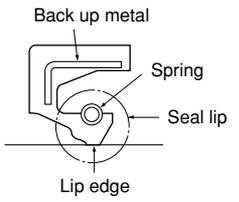
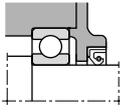
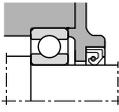
- **Contact seals:** A contact seal is a seal whereby a

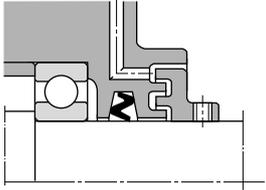
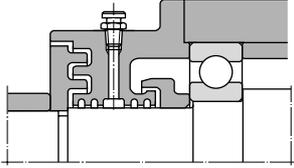
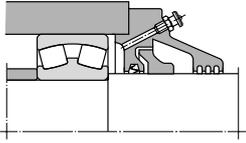
formed synthetic rubber lip on a steel plate is pressed against the shaft. Contact seals are generally far superior to non-contact seals in sealing efficiency, although their friction torque and temperature rise coefficients are higher. Furthermore, because the lip portion of a contact seal slides while in contact with the shaft, the allowable seal peripheral speed varies depending on seal type.

Lubrication is required in the place where the seal lip makes contact with the shaft. Ordinary bearing lubricant can also be used for this purpose.

The following chart lists the special characteristics of seals and other points to be considered when choosing an appropriate seal.

| Type | Seal construction | Name | Seal characteristics and selection considerations | | | | | | | | | | | | | | | | | |
|-------------------|-------------------|---|---|-------------------|--------------|----------|---------|-------------|---------|-------------------|--------------|--|------------------|-----------------|-------|---------|---------|--------|---------|---------|
| Non-contact seals | | Clearance seal | This is an extremely simple seal design with a small radial clearance. | | | | | | | | | | | | | | | | | |
| | | Oil groove seal (oil grooves on housing side) | Several concentric oil grooves are provided on the housing inner diameter to greatly improve the sealing effect. When the grooves are filled with lubricant, the intrusion of contaminants from the outside is prevented. | | | | | | | | | | | | | | | | | |
| | | Oil groove seal (oil grooves on shaft and housing side) | Oil grooves are provided on both the shaft outer diameter and housing inner diameter for a seal with even greater sealing efficiency. | | | | | | | | | | | | | | | | | |
| | | Axial labyrinth seal | This seal has a labyrinth passageway on the axial side of the housing. | | | | | | | | | | | | | | | | | |
| | | Radial labyrinth seal | A labyrinth passageway is affixed to the radial side of the housing. For use with split housings. This offers better sealing efficiency than axial labyrinth seals. | | | | | | | | | | | | | | | | | |
| | | Aligning labyrinth seal | The seal's labyrinth passageway is slanted and has sufficient clearance to prevent contact between the housing projections and the shaft even as the shaft realigns. | | | | | | | | | | | | | | | | | |
| | | | <p>Cautionary points regarding selection</p> <ul style="list-style-type: none"> • In order to improve sealing efficiency, clearances between the shaft and housing should be minimized. However, care should be taken to confirm shaft/bearing rigidity and other factors to avoid direct shaft-housing contact during operation. <p>Oil groove clearance (reference)</p> <table border="1"> <thead> <tr> <th>Shaft diameter mm</th> <th>Clearance mm</th> </tr> </thead> <tbody> <tr> <td>Up to 50</td> <td>0.2~0.4</td> </tr> <tr> <td>50 or above</td> <td>0.5~1.0</td> </tr> </tbody> </table> <ul style="list-style-type: none"> • Oil groove width, depth (reference) width : 2~5 mm depth : 4~5 mm • Three or more oil grooves should be provided. • Sealing efficiency can be further improved by filling the oil groove portion with grease of which the consistency grade is 150 to 200. • Grease is generally used as the lubricant for labyrinth seals, and, except in low speed applications, is commonly used together with other sealing devices. <p>Cautionary points regarding selection</p> <ul style="list-style-type: none"> • In order to improve sealing efficiency, labyrinth passageway clearances should be minimized. However, care should be taken to confirm shaft/bearing rigidity, fit, internal clearances and other factors to avoid direct contact between labyrinth projections during operation. <p>Labyrinth clearance (reference)</p> <table border="1"> <thead> <tr> <th rowspan="2">Shaft diameter mm</th> <th colspan="2">Clearance mm</th> </tr> <tr> <th>Radial direction</th> <th>Axial direction</th> </tr> </thead> <tbody> <tr> <td>—~ 50</td> <td>0.2~0.4</td> <td>1.0~2.0</td> </tr> <tr> <td>50~200</td> <td>0.5~1.0</td> <td>3.0~5.0</td> </tr> </tbody> </table> <ul style="list-style-type: none"> • Sealing efficiency can be further improved by filling the labyrinth passageway with grease of which the consistency grade is 150 to 200. • Labyrinth seals are suitable for high speed applications. | Shaft diameter mm | Clearance mm | Up to 50 | 0.2~0.4 | 50 or above | 0.5~1.0 | Shaft diameter mm | Clearance mm | | Radial direction | Axial direction | —~ 50 | 0.2~0.4 | 1.0~2.0 | 50~200 | 0.5~1.0 | 3.0~5.0 |
| Shaft diameter mm | Clearance mm | | | | | | | | | | | | | | | | | | | |
| Up to 50 | 0.2~0.4 | | | | | | | | | | | | | | | | | | | |
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| Shaft diameter mm | Clearance mm | | | | | | | | | | | | | | | | | | | |
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| Type | Seal construction | Name | Seal characteristics and selection considerations | | | | | | | | | | | | | | | | | | | | | |
|----------------------|---|---|--|----------------------|---|-------------------|----------------|------------|------------|-----------|----------------|------------|------------|--------------------|------------|------------|--------|----------------|--|------------------|--|-------------------|---|------------|
| Non-contact seals |  | Oil comb sleeve | In this design, lubricating oil that makes its way out of the housing along the shaft is thrown off by projections on the oil comb sleeve and recirculated. | | | | | | | | | | | | | | | | | | | | | |
| |  | Slinger provided in the housing | Seal type whereby a slinger is provided in the housing that prevents lubricant from leaking by centrifugal force produced by rotation. | | | | | | | | | | | | | | | | | | | | | |
| |  | Slinger provided outside the housing | By mounting a slinger on the outside of the housing, centrifugal force helps to prevent dust and other solid contaminants from entering. | | | | | | | | | | | | | | | | | | | | | |
| Contact seals |  | Z grease seal | In cross section resembling the letter "Z," this seal's empty spaces are filled with grease. The seal is commonly used with a plummer block (bearing housing). | | | | | | | | | | | | | | | | | | | | | |
| |  | V-ring seal | This design enhances sealing efficiency with a lip that seals from the axial direction. With the aid of centrifugal force, this seal also offers effective protection against dust, water, and other contaminants entering the bearing. Can be used for both oil and grease lubrication. At seal peripheral speeds in excess of 12 m/s, seal ring fit is lost due to centrifugal force, and a clamping band is necessary to hold it in place. | | | | | | | | | | | | | | | | | | | | | |
| |  | Oil seal | Oil seals are widely used, and their shapes and dimensions are standardized under JIS B 2402. In this design, a ring-shaped spring is installed in the lip section. As a result, optimal contact pressure is exerted between the lip edge and shaft surface, and sealing efficiency is good. When the bearing and oil seal are in close proximity, the internal clearance of the bearing may be reduced by heat produced by the oil seal. In addition to considering the heat generated by contact seals at various peripheral speeds, internal bearing clearances must also be selected with caution. Depending on its orientation, the seal may function to prevent lubricant from leaking out or foreign matter from getting in. | | | | | | | | | | | | | | | | | | | | | |
| |  | For dust proof | | | | | | | | | | | | | | | | | | | | | | |
| |  | For preventing lubricant leakage | | | | | | | | | | | | | | | | | | | | | | |
| | | | <p>Cautionary points regarding selection</p> <ul style="list-style-type: none"> Seal type whereby a slinger that utilizes centrifugal force is provided on the rotating shaft. If mounted on the inside of the housing, the slinger should function to seal in lubricant by centrifugal force produced by rotation. If mounted on the outside of the housing, the slinger should function to seal out foreign matter by fan effect produced by rotation. These seal types are commonly employed together with other sealing devices. | | | | | | | | | | | | | | | | | | | | | |
| | | | <p>Cautionary points regarding selection</p> <p>Shaft surface roughness (reference)</p> <table border="1"> <thead> <tr> <th rowspan="2">Peripheral speed m/s</th> <th colspan="2">Surface roughness</th> </tr> <tr> <th>Ra</th> <th>Rmax</th> </tr> </thead> <tbody> <tr> <td>~ 5</td> <td>0.8a</td> <td>3.2s</td> </tr> <tr> <td>5~10</td> <td>0.4a</td> <td>1.6s</td> </tr> <tr> <td>10~</td> <td>0.2a</td> <td>0.8s</td> </tr> </tbody> </table> <p>Shaft material (reference)</p> <table border="1"> <tbody> <tr> <td>Material</td> <td>Machine structural carbon steel, Low carbon alloy steel, Stainless steel</td> </tr> <tr> <td>Surface hardness</td> <td>HRC 40 or higher necessary HRC 55 or higher advisable</td> </tr> <tr> <td>Processing method</td> <td>Final grinding without repeat (moving), or buffed after hard chrome plating</td> </tr> </tbody> </table> | Peripheral speed m/s | Surface roughness | | Ra | Rmax | ~ 5 | 0.8a | 3.2s | 5~10 | 0.4a | 1.6s | 10~ | 0.2a | 0.8s | Material | Machine structural carbon steel, Low carbon alloy steel, Stainless steel | Surface hardness | HRC 40 or higher necessary HRC 55 or higher advisable | Processing method | Final grinding without repeat (moving), or buffed after hard chrome plating | |
| Peripheral speed m/s | Surface roughness | | | | | | | | | | | | | | | | | | | | | | | |
| | Ra | Rmax | | | | | | | | | | | | | | | | | | | | | | |
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| 10~ | 0.2a | 0.8s | | | | | | | | | | | | | | | | | | | | | | |
| Material | Machine structural carbon steel, Low carbon alloy steel, Stainless steel | | | | | | | | | | | | | | | | | | | | | | | |
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| | | | <p>Allowable speed/temperature according to seal type/material (reference)</p> <table border="1"> <thead> <tr> <th>Seal type/material</th> <th>Allowable peripheral speed m/s ($v(m/s) = \frac{\pi \times d(mm) \times n(r/min)}{60,000}$)</th> <th>Allowable temp °C</th> </tr> </thead> <tbody> <tr> <td>Nitrile rubber</td> <td>16 or less</td> <td>-25 ~ +120</td> </tr> <tr> <td rowspan="2">Oil seals</td> <td>Acrylic rubber</td> <td>26 or less</td> <td>-15 ~ +150</td> </tr> <tr> <td>Fluorinated rubber</td> <td>32 or less</td> <td>-30 ~ +200</td> </tr> <tr> <td>Z-seal</td> <td>Nitrile rubber</td> <td>6 or less</td> <td>-25 ~ +120</td> </tr> <tr> <td>V-ring</td> <td>Nitrile rubber</td> <td>40 or less</td> <td>-25 ~ +120</td> </tr> </tbody> </table> | Seal type/material | Allowable peripheral speed m/s ($v(m/s) = \frac{\pi \times d(mm) \times n(r/min)}{60,000}$) | Allowable temp °C | Nitrile rubber | 16 or less | -25 ~ +120 | Oil seals | Acrylic rubber | 26 or less | -15 ~ +150 | Fluorinated rubber | 32 or less | -30 ~ +200 | Z-seal | Nitrile rubber | 6 or less | -25 ~ +120 | V-ring | Nitrile rubber | 40 or less | -25 ~ +120 |
| Seal type/material | Allowable peripheral speed m/s ($v(m/s) = \frac{\pi \times d(mm) \times n(r/min)}{60,000}$) | Allowable temp °C | | | | | | | | | | | | | | | | | | | | | | |
| Nitrile rubber | 16 or less | -25 ~ +120 | | | | | | | | | | | | | | | | | | | | | | |
| Oil seals | Acrylic rubber | 26 or less | -15 ~ +150 | | | | | | | | | | | | | | | | | | | | | |
| | Fluorinated rubber | 32 or less | -30 ~ +200 | | | | | | | | | | | | | | | | | | | | | |
| Z-seal | Nitrile rubber | 6 or less | -25 ~ +120 | | | | | | | | | | | | | | | | | | | | | |
| V-ring | Nitrile rubber | 40 or less | -25 ~ +120 | | | | | | | | | | | | | | | | | | | | | |

| Type | Seal construction | Name | Seal characteristics and selection considerations |
|-------------------|---|---|---|
| Combination seals |  | Z-seal + Labyrinth seal | This is an example of an axial labyrinth seal which has been combined with a Z-seal to increase its sealing efficiency. The axial labyrinth seal is affixed to the shaft with a setting bolt or other method. In the diagram on the left, both the direction of the Z-seal and the labyrinth seal are oriented to keep dust and other contaminants out of the bearing. Because a Z-seal has been incorporated, the allowable peripheral speed should not exceed 6 m/s. |
| |  | Labyrinth seal + Oil groove seal + Slinger | This is an example of a combination of three different non-contact seals. It has the advantage of preventing both lubricant leakage from inside the bearing and infiltration of dust and other contaminants from the outside. It is widely used on mining equipment and as a sealing system with plummer blocks in extremely dusty application conditions. |
| |  | Oil groove seal + Slinger + Z-seal | This is an example where an oil groove seal and slinger have been combined with a Z-seal to increase its sealing efficiency. In the diagram on the left, all three seals have been oriented to keep dust and other contaminants out of the bearing. The combination is widely used on mining equipment and as a sealing system with plummer blocks in extremely dusty application conditions. |